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**Content Based  
Chemical Engineering**

CONTROL VALVES - find flow, pressures known

**BASIS:** Control Valve Key Equations, ISA-75.01-1985 (R1995) formerly ISA-S75.01-1985 (R1995)

**NOTE:** Always begin a new case by retrieving the original file. Direct entry of data in cells that originally contain table lookups could cause functions to be lost, or incorrect calculations. I format cells requiring entry colored **RED**; calculated values are black.

- 1.) Select the appropriate worksheet for the calculation needed.
- 2.) Enter instrument (control valve) identification at [C4].
- 3.) Enter fluid by using [=], then going to fluid name in the gas or liquid tables, and double-clicking the mouse or high-lighting and hitting ther [return key]. Fluid specifics will be returned via lookup tables. For fluids NOT in my tables enter values in the appropriate cells.
- 4.) Enter Cv values as appropriate for Process Safety Calculations these values should be the 100% open value as found in the control valve specification or from the vendors literature.

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Consistent with GOOD ENGINEERING PRACTICE, the burden rests with the USER of these spreadsheets to review ALL calculations, and assumptions. The USER IS FULLY RESPONSIBLE for the results or decisions based on calculations.

This Spreadsheet Requires MACROS to be ENABLED to ASSURE proper operation. See the Workbook Help Sheet for Additional Instructions on Use.

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ISA - Control Valve - Compressible Fluid; Flow of Gas or Vapor
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VALVE : 1-1/4" FCV  
 VALVE TYPE : Ball / Standard Port (dia app 0.8d) - Either  
 FLUID : STEAM

$C_v = 47$	$M_w = 18.02$ mol. weight
valve ID = 1.250 in	$k = 1.314451$
pipe ID = 2.000 in	$v_1 = 11.996933$ ft <sup>3</sup> /lb
P1 - in = 20 psig	$P_{1a} = 34.7$ psia
P2 - out = 16.5 psig	$P_{2a} = 31.2$ psia
temp. = 258.25991 °F	$T_{1a} = 718.3$ °R
$\Delta P = 3.5$ psi	$Z = 1.000$

$$W = 19.3 \cdot F_p \cdot C_v \cdot P_{1a} \cdot Y \cdot \left( \frac{\Delta P}{P_{1a}} \cdot \frac{M_w}{T_{1a}} \cdot Z \right)^{0.5}$$

... Determine  $F_p$  ...

$$F_p = \left[ \left( \sum K \cdot C_v^2 / 890 \cdot d^4 \right) + 1 \right]^{-1/2}$$

reducer, $K_1 = 0.424$
increaser, $K_2 = 0.371$
$\Sigma K = 0.795$

$F_p = 0.904$

... Determine  $Y$  ...

$$Y = 1 - x / (3 \cdot F_k \cdot x_T \text{ ..or.. } x_{Tf})$$

$x_t = 0.1500$
$x_{TP} = 0.1367$

$Y = 0.737955$

... Calculation ...

$$W = 19.3 \cdot F_p \cdot C_v \cdot P_{1a} \cdot Y \cdot \left( \frac{\Delta P}{P_{1a}} \cdot \frac{M_w}{T_{1a}} \cdot Z \right)^{0.5}$$

$W = 1056$  lb/hr of steam

ISA - Control Valve - Compressible Fluid; Flow of Gas or Vapor
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VALVE : 32-mm FCV  
 VALVE TYPE : Ball / Standard Port (dia app 0.8d) - Either  
 FLUID : STEAM

$C_v = 47$	$M_w = 18.02$ mol. weight
valve ID = 31.75 mm	$k = 1.3$
pipe ID = 50.8 mm	$v_1 = 0.7490489$ m <sup>3</sup> /kg
P1 - in = 1.378952 barG	$P_{1a} = 2.378952$ barA
P2 - out = 1.137635 barG	$P_{2a} = 2.137635$ barA
temp. = 125.69499 °C	$T_{1a} = 398.8$ °K
$\Delta P = 0.241317$ bar	$Z = 1.000$

$$W = 94.8 \cdot F_p \cdot C_v \cdot P_{1a} \cdot Y \cdot (\Delta P / P_{1a} \cdot M_w / T_{1a} \cdot Z)^{0.5}$$

... Determine  $F_p$  ...

$$F_p = [(\Sigma K \cdot C_v^2 / 0.00214 \cdot d^4) + 1]^{-1/2}$$

reducer, $K_1 = 0.424$
increaser, $K_2 = 0.371$
$\Sigma K = 0.795$

$F_p = 0.904$

... Determine  $Y$  ...

$$Y = 1 - x / (3 \cdot F_k \cdot x_T \text{ ..or.. } x_{Tf})$$

$x_T = 0.1500$
$x_{TP} = 0.1368$

$Y = 0.733732$

... Calculation ...

$$W = 94.8 \cdot F_p \cdot C_v \cdot P_{1a} \cdot Y \cdot (\Delta P / P_{1a} \cdot M_w / T_{1a} \cdot Z)^{0.5}$$

$W = 476$  kg/hr of steam

ISA - Control Valve - Incompressible Fluid; Flow of Nonvaporizing Liquid

VALVE : FV-5501; DEAERATOR FEEDWATER  
 VALVE TYPE : Globe Single Port / Characterized Cage - Open  
 FLUID : WATER

valve ID =	1.250	in	C <sub>v</sub> =	20
pipe ID =	2.000	in	γ <sub>1</sub> =	57.42073 lb/ft <sup>3</sup>
P1 - in =	20	psig	P1 <sub>a</sub> =	34.7 psia
P2 - out =	16.5	psig	P2 <sub>a</sub> =	31.2 psia
ΔP =	3.5	psi	v =	0.2008491 centistokes
			temp. =	298 °F

... Determine F<sub>p</sub> ...

$$F_p = [(\Sigma K \cdot C_v^2 / 890 \cdot d^4) + 1]^{-1/2}$$

reducer, K <sub>1</sub> =	0.424
increaser, K <sub>2</sub> =	0.371
ΣK =	0.795

F<sub>p</sub> = 0.4361633

... Determine Re<sub>v</sub>, F<sub>r</sub>, & q<sub>t</sub> ...

F <sub>d</sub> =	1.00	Re <sub>v</sub> =	819,884
F <sub>L</sub> =	0.90	F <sub>r</sub> =	1.00
		q <sub>t</sub> =	39.01

... Predicted Flow Rate ...

$$W = F_r \cdot q_t$$

W = 39.005151 gpm

ISA - Control Valve - Incompressible Fluid; Flow of Nonvaporizing Liquid

VALVE : FV-5501; DEAERATOR FEEDWATER  
 VALVE TYPE : Globe Single Port / Characterized Cage - Open  
 FLUID : WATER

valve ID = 31.75 mm	C <sub>v</sub> = 20
pipe ID = 50.8 mm	γ <sub>1</sub> = 920.204 kg/m <sup>3</sup>
P1 - in = 1.378952 barG	P1 <sub>a</sub> = 2.378952 barA
P2 - out = 1.137635 barG	P2 <sub>a</sub> = 2.137635 barA
ΔP = 0.241317 bar	v = 0.2008491 centistokes
temp. = 147.77778 °C	temp. = 298 °F

... Determine F<sub>p</sub> ...

$$F_p = [(\Sigma K \cdot C_v^2 / 0.00214 \cdot d^4) + 1]^{-1/2}$$

reducer, K <sub>1</sub> = 0.424	
increaser, K <sub>2</sub> = 0.371	
	ΣK = 0.795

F<sub>p</sub> = 0.4362093

... Determine Re<sub>v</sub>, F<sub>r</sub>, & q<sub>t</sub> ...

F <sub>d</sub> = 1.00	Re <sub>v</sub> = 819,779
F <sub>L</sub> = 0.90	F <sub>r</sub> = 1.00
	q <sub>t</sub> = 39.00

... Predicted Flow Rate ...

$$W = F_r \cdot q_t \cdot 0.227$$

W = 8.85 m<sup>3</sup>/hr

ISA - Control Valve - Incompressible Fluid; Choked Flow of Vaporizing Liquid

VALVE : PCV-950008; DESUPERHEAT SPRAY FROM FEEDWATER  
 VALVE TYPE : Angle / Characterized Cage - Close  
 FLUID : Water

valve ID =	4.000	in	C <sub>v</sub> =	90	
pipe ID =	6.000	in	G <sub>f</sub> =	0.998	sp. wt.
P1 - in =	200	psig	P1 <sub>a</sub> =	214.7	psia
P2 - out =	124	psig	P2 <sub>a</sub> =	138.7	psia
ΔP =	76	psi	temp. =	70	°F

... Determine F<sub>LP</sub> ...

F<sub>L</sub> = 0.80                      inlet reducer, K<sub>i</sub> = 0.401

$$F_{LP} = F_L \cdot [(K_i \cdot F_L^2 \cdot C_v^2 / 890 \cdot d^4) + 1]^{-1/2}$$

$$F_{LP} = 0.3963814$$

... Determine Re<sub>v</sub>, F<sub>r</sub>, & q<sub>t</sub> ...

p<sub>v</sub> = 0.36      psia                                      p<sub>c</sub> = 3198.72

F<sub>F</sub> = 0.96    p<sub>vc</sub> = 0.344107

... Predicted Flow Rate ...      q<sub>max</sub> = 1.00 · F<sub>LP</sub> · C<sub>v</sub> · [(p<sub>1</sub> - p<sub>vc</sub>) / G<sub>f</sub>]<sup>0.5</sup>

$$q_{max} = 522.8272 \text{ gpm}$$

ISA - Control Valve - Incompressible Fluid; Choked Flow of Vaporizing Liquid

VALVE : PCV-950008; DESUPERHEAT SPRAY FROM FEEDWATER  
 VALVE TYPE : Angle / Characterized Cage - Close  
 FLUID : Water

valve ID = 101.600 mm	C <sub>v</sub> = 90
pipe ID = 152.400 mm	G <sub>f</sub> = 0.998 sp. wt.
P1 - in = 13.7931 barG	P1 <sub>a</sub> = 14.8 barA
P2 - out = 8.55172 barG	P2 <sub>a</sub> = 9.6 barA
ΔP = 5.24138 barG	temp. = 21.11 °C

... Determine F<sub>LP</sub> ...

F<sub>L</sub> = 0.80                      inlet reducer, K<sub>i</sub> = 0.401

$$F_{LP} = F_L \cdot \left[ \left( K_i \cdot F_L^2 \cdot C_v^2 / 0.00214 \cdot d^4 \right) + 1 \right]^{-1/2}$$

F<sub>LP</sub> = 0.3963843

... Determine Re<sub>v</sub>, F<sub>r</sub>, & q<sub>t</sub> ...

p<sub>v</sub> = 18.59 mmHg                      p<sub>c</sub> = 3198.72

F<sub>F</sub> = 0.96                                      p<sub>vc</sub> = 0.344083

... Predicted Flow Rate ...

$$q_{\max} = 0.865 \cdot F_{LP} \cdot C_v \cdot \left[ (p_1 - p_{vc}) / G_f \right]^{0.5}$$

q<sub>max</sub> = 117.4703 m<sup>3</sup>/hr